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# Ultrafast, miniature soft actuators

To cite this article: Onur Bas et al 2021 Multifunct. Mater. 4 045001

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## **Multifunctional Materials**

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**RECEIVED** 21 June 2021

REVISED 30 September 2021

ACCEPTED FOR PUBLICATION 13 October 2021

PUBLISHED 1 November 2021

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**Keywords:** soft robotics, actuators, composites, fibers, inflatable, fast Supplementary material for this article is available online

Ultrafast, miniature soft actuators

#### Abstract

PAPER

The quest for an advanced soft robotic actuator technology that is fast and can execute a wide range of application-specific tasks at multiple length scales is still ongoing. Here, we demonstrate a new design and manufacturing strategy that leads to high-speed inflatable actuators exhibiting diverse movements. Our approach leverages the concept of miniaturisation to reduce the required volume of fluid for actuation as well as fibre-reinforcement to improve the efficiency of actuators in converting delivered fluids into fast and predictable movement. To fabricate the designs, we employ a class of additive manufacturing technology called melt electrowriting. We demonstrate 3D printing of microfibre architectures on soft elastomers with precision at unprecedently small length scales, leading to miniaturised composite actuators with highly controlled deformation characteristics. We show that owing to their small dimensions and deterministically designed fibrous networks, our actuators require extremely low amounts of fluid to inflate. We demonstrate that actuators with a length of 10–15 mm and an inner diameter 1 mm can reach their full range of motion within  $\sim 20$  ms without exploiting snapping instabilities or material non-linearities. We display the speed of our actuators by building an ultrafast, soft flycatcher. We also show that our actuators outperform their counterparts with respect to achievable movement diversity and complexity.

## 1. Introduction

The field of soft robotics proposed the development of versatile actuators as one of the first marks on its roadmap to next-generation robotics, due to the integral role of these technologies in the function and performance of robots [1–3]. It is known that actuators determine the size, cost, power source, control mechanism, and general design of any robotic system [4], therefore strong emphasis has been placed on the advancement of these components. Although a wide range of actuator designs have been proposed, due to their ease of operation, low cost and ability to achieve a large range of deformations, inflatable actuators have been the primary subject of interest [5]. However, these actuators are often considered too slow for most applications as their speed is limited by large inflation volumes that are needed to create deformations and the consequent large viscous forces needed to let this volume pass through narrow tubes. Several design strategies leveraging snapping instabilities [6], material non-linearity [7], stored elastic energy [8], as well as explosive chemical reactions [9] have been proposed to overcome this limitation. However, the functionality of the resulting inflatable actuators is highly limited as very specific considerations have to be taken into account in their design and manufacturing. Combining high actuation speeds without compromising functionality has been a major challenge and a high-speed inflatable actuator concept that can execute an

extensive range of application-specific tasks is yet to be demonstrated. In this work, we aim at developing highly functional and modular actuators that achieve high speeds.

#### 2. Results and discussions

To accomplish this goal, we investigate mechanisms that lay at the basis of nature's fast-acting structures that rely on the transportation of fluids for actuation. It has been shown that these fluid-driven actuators harness miniaturisation to achieve fast movements [10]. For instance, the Aldrovanda can close its leaves in 20 ms, which is ~10 times more rapidly than the Venus flytrap, due to its significantly smaller size (approximately 1/10th of Venus flytrap) [10]. Miniaturised designs influence both the required volume of fluid ( $\Delta_{Volume}$ ) that is needed to be displaced, as well as the distance it needs to travel, enabling rapid movements. Thus, we hypothesise that an analogues miniaturisation strategy can be applied to inflatable actuators (see figure 1(A) for the schematic illustration of the concept).

In addition to miniaturisation, it is essential to improve the efficiency of actuators in converting delivered fluids into fast and predictable movement. As the transportation of fluids is a challenge in these systems, available volume influx has to be efficiently converted towards desired deformations by avoiding unnecessary volumetric expansions that do not contribute to the overall desired deformation (figure 1(A)). To determine how design can influence this efficiency, we performed finite element (FE) simulations (see supplementary text for further information available online at stacks.iop.org/MFM/4/045001/mmedia) on three common bending actuator designs (eccentric [11], corrugated membrane [12] and fibre-reinforced [13]) and measured the inflation volume required to obtain the same degree of bending. To eliminate the effect of size, we normalised the inflation volume by  $L^3$ , where we have taken the length 'L' of the actuator as the characteristic length scale. From this analysis, we verify that the design of actuators has a major influence on bending performance. The same bending angle is reached at lower input volume for those designs that limit parasitic expansion and cross-sectional deformations. As shown in figure 1(B), the actuator with an eccentric design [14] first has to be inflated substantially to start bending. Then, more than 5 times  $\Delta_{Volume}$  has to be delivered to achieve 90° bending when compared to the fibre-reinforced actuator design. Similarly, the formation of major bulges can be seen in the corrugated membrane actuator design, which is the hallmark of compromised performance. Based on these findings, we conclude that deterministically designed high-modulus fibre architectures are effective in converting fluid volume influx into the desired deformation. In addition to their efficiency, which is largely overlooked and yet to be exploited in the literature, fibre-reinforced actuators are proven to be highly versatile with their ability to achieve tailored motions [13, 15], making them the ideal choice in our application.

While the previous analysis pointed that fibre reinforced design has the highest potential to achieve high speeds, it did not provide any information regarding the influence of the materials and/or geometry on the performance of the actuators. To investigate the dependency of inflation volume on these two factors, we conducted a series of FE analyses. We designed and simulated a fibre reinforced bending actuator consisting of double-helical fibres (both clockwise and counterclockwise) with a pitch of 390  $\mu$ m as well as three longitudinally placed fibres on one side that create bending, surrounding a hollow cylinder with an internal diameter of 1 mm, a wall thickness of 0.2 mm and length of 10 mm. In our simulations (which were conducted using the commercial code Abaqus/Standard), we discretised the cylinder using second-order hybrid tetrahedral elements (element type: C3D10H and the fibres with three-node quadratic beam elements (element type: B32) and actuated the models supplying incompressible fluid to the internal cavity via a fluid-cavity interaction. We modelled the material of the cylinder as incompressible neo-Hookean with the initial shear modulus  $\mu_0$ , while the fibres are assumed to be linear elastic with Young's modulus E and Poisson's ratio of 0.3. To begin with, we considered  $\mu_0 = 15$  kPa and E = 160 MPa and found that both the bending angle and pressure vary more or less linear with inflation volume (see figure 2). From the graphs of figure 2(A), we can identify three actuator characteristics: the inflation volume that is needed to achieve a bending angle of 90° ( $\Delta V_{90}$ ), the corresponding pressure ( $p_{90}$ ) and the needed energy ( $\Delta E_{90}$ ), which can be calculated as the area under the pressure–volume curve. For this particular actuator,  $\Delta V_{90} = 2.35 \ \mu$ l,  $p_{90} = 5.4$  kPa and  $\Delta E_{90} = 0.0064$  mJ. Next, we tested different initial shear moduli values ranging between 7.5 kPa <  $\mu_0$  < 120 kPa for the elastomeric matrix and Young's moduli values ranging between 80 Mpa < E < 1.28 Gpa for the fibres. As these two parameters directly influence the bending stiffness of either the cylinder (EI =  $3\mu_0 \times \pi/4 \cdot (r_0^4 - r_i^4)$ ), with  $r_0$  and  $r_i$  being the inner and outer radius of the cylinder) and the fibres (EI =  $E \times \pi/4 \times r^4$ , with *r* being the fibre radius), we can use them to assess the influence of both geometrical and material parameters on the performance of the actuators. In figures 2(B)–(D), we display these influences by plotting respectively  $\Delta V_{90}$ ,  $p_{90}$  and  $\Delta E_{90}$  for different bending stiffnesses of both the cylinder or the surrounding fibres, where stars indicate the previously reported values. Regarding the pressure that is needed to achieve a  $90^{\circ}$  bending angle (figure 2(B)), we can



**Figure 1.** Design rationale for achieving high-speed soft actuators. (A) Schematic illustration demonstrating the concept of fibre reinforcement and miniaturisation to reduce  $\Delta_{\text{Volume}}$  needed for actuation, which is critical for achieving high-speed actuation via regular pressure suppliers. (B) The influence of actuator design on the performance (capability of converting volume inputs into desired deformations). The figure indicates that certain actuator designs exhibit less parasitic deformations when inflated and require less volume input to achieve the same degree of bending. The dotted line depicts the relative inflation volume that is needed to achieve a bending angle of 90° for each actuator.

conclude that a higher pressure is needed when the bending stiffness of the cylinder increases. Surprisingly, the bending stiffness of the fibre does not have a large influence on the required input pressure levels. Regarding the inflation volume (figure 2(C)), we found that a combination of a stiff tube with compliant fibres and a combination of stiff fibres with a compliant tube lead to higher inflation volumes than when both are stiff or compliant. We can thus conclude that there is an optimal ratio of bending stiffnesses, where fibres need to be  $\sim 15$  times stiffer than the elastomeric matrix, which has been indicated by a dotted line on the figure. Lastly, The energy that is needed to bend 90° (figure 2(D)) is dominated by and follows the same trends as the pressure dependency, which is logical since the relative pressure variation is larger than the relative volume variation while varying stiffnesses.

To create these highly dynamic bending actuators with an optimal fibre composition, we cannot rely on production processes described in the literature, as they either are incompatible with small scale production processes [13, 15–17], or lack control over fibre placements [18–20]. Therefore we developed a new manufacturing strategy that facilitates the fabrication of miniaturised composite soft actuators with precision at small length scales. We use melt electrowriting (MEW) technology [21, 22], a class of additive manufacturing system, which combines the capability of electrospinning systems to produce ultra-fine fibres (fibre diameters between 1 and 50  $\mu$ m) with the design freedom of 3D printing. In this automated process, we apply a thin layer of uncured soft silicone-based elastomer on the rotational collector of our MEW system. By using a rod integrated into our MEW device, we move the stage in *x*-direction back and forwards with an elastomer and create a thin, uniform layer of the silicone tube to achieve enhanced bonding between the fibres and matrix material (see figure 3 and movie S1 for the schematic illustration and video of the fabrication process, respectively). After the completion of the 3D printing process, we allow the silicone to fully cure and connect the actuators to a pressure source after sealing their tip.



**Figure 2.** Parameter variation study. (A) Simulated action characteristics for an actuator composed out of rubber with  $\mu_0 = 15$  kPa and fibres with E = 160 MPa. (B) The pressure that is needed to achieve a bending angle of 90° for changing bending stiffnesses of the fibres and the elastomeric tube. (C) The inflation volume that is needed to achieve a bending angle of 90° for changing bending stiffnesses of the fibres and the elastomeric tube. The dotted line depicts optimal actuators that require a minimum in inflation volume. (D) The energy that is needed to achieve a bending angle of 90° for changing bending stiffnesses of the fibres and the elastomeric tube. The dotted line depicts optimal actuators that require a minimum in inflation volume. (D) The energy that is needed to achieve a bending angle of 90° for changing bending stiffnesses of the fibres and the elastomeric tube. The stars are indicative of the performance of the actuator displayed in (A).

As demonstrated in figure 4, we successfully fabricated a miniaturised bending actuator with an internal void diameter of 1 mm, length of 10 mm and a wall thickness of 0.2 mm using our manufacturing technique (see figure S1 for the technical drawing). The scanning electron micrograph (figure 4(A)) shows the accurate placement of the fibres as well as their good continuity and consistency (see table S1 for the detailed characterisation of the dimensions of the fabricated actuator). We selected the constituent materials of this actuator in accordance with the established principles depicted in figure 2. As our findings indicate that the use of soft matrix materials reduces the required actuation energy, we applied the softest grade silicone within the product family of a widely used elastomer (Ecoflex with a shore hardness of 00-10). For the fabrication of the fibre phase, we preferred polycaprolactone due to its excellent rheological properties and processability via MEW process as well as the favourable mechanical properties (elastic modulus of 320 MPa), leading to an actuator with a fibre-to-matrix bending stiffness ratio of  $\sim$ 7.5. Although this ratio is smaller than the identified ideal ratio, our simulations suggest that this actuator (internal diameter of 1 mm, and length of 10 mm) require  $\Delta_{\text{Volume}}$  of 2.3559  $\mu$ l to achieve 90° bending, which is marginally higher than that of an actuator built with materials having a bending stiffness ratio of 15 ( $\Delta_{\text{Volume}}$  of 2.3143  $\mu$ l). Overall, this material combination yielded high performant bending actuators that are also easy to manufacture, handle and characterise as demonstrated in figure 4.

After pressurizing the actuator, we indeed observed the intended large bending movements with minimum parasitic deformations (see figure 4(A)) (see figure S5 for a bending actuator without helical fibres exhibiting large parasitic deformations). Figure 4(B) shows the magnitude of the deformations achieved by the actuator at given air pressure both experimentally and as computed by means of FE modelling (see movie S2). By downscaling the dimensions to a diameter of 1 mm, we were able to fabricate bending actuators that reach full stroke  $(270^\circ)$  when inputting only volumes of less than 7.5  $\mu$ l. Furthermore, by modelling the





volumetric expansion of our actuator using finite element modelling (FEM), we see radially restricted actuators are characterised by a linear displacement–volume relationship. This means that the input volume is efficiently redirected towards only one spatial dimension, giving a leveraging effect for fast actuation. To confirm this, we have tested our bending actuators under a high-frequency pneumatic input (on-off), where the input air pressure was adjusted such that a full stroke was reached at the end of the cycle (see figures 4(C) and figure S7 for details). We were able to achieve an actuation frequency reaching 30 Hz, where complete bending and recovery to the initial state takes place within  $\sim$ 30 ms (see movie S7). Further, we see that the dynamics are limited by the deflation part of the cycle. In contrast to inflation where we can adjust the input pressure to reach full stroke quicker, the deflation of the actuator is limited by an atmospheric back-pressure, resulting in a maximum actuation frequency of 30 Hz.

The presented methodology of combining miniaturisation with a fibre reinforcement design showed to be a highly successful pathway of creating highly dynamic actuators. However, this methodology is not limited to only bending deformations. Using the fabrication freedom of additive manufacturing, we can



**Figure 4.** Characterisation of the performance of bending actuators. (A) Deformation behaviour of a representative bending actuator in comparison to the deformations obtained via simulations. Scanning electron microscopy image of the actuator is also shown. (B) Graphs of inflation volume vs deformation and input pressure vs deformation obtained via experiments (n = 3) and simulations. (C) Air pressure values measured while characterising the actuation performance of the bending actuators at different actuation frequencies (5, 10, 20–30 Hz). At frequencies >~10 Hz, pressure values exceeding the regular actuation pressure requirements of the actuators were applied to be able to reach the desired pressure levels within each actuation cycle (see figure S7 for exact values). The duty cycle was kept constant (on-to-off duration ratio of 2) for all the experiments.





deposit fibres at arbitrary positions and orientations, as displayed in figure 5. The fibre architectures that give rise to the unique deformation of these actuators are displayed using scanning electron microscope (SEM) images, where for twining we combined helical fibres with three grouped eccentric fibres, for extending we remove the eccentric fibres, and for contracting we only use eccentric fibres that are evenly spaced (see movies S3-5). Further, these actuators were dynamically tested, resulting in a maximum actuation frequency of 20 Hz for twining, 30 Hz for contraction and 30 Hz for elongation actuators (see movies S8–S10 for high-speed actuation videos, figures S2–S4 for technical drawings of the actuators and figure S6 for detailed characterisation results).

To benchmark these actuators, we have collected the dynamic data (time to complete an actuation cycle) of inflatable actuators from the literature and plotted it against their length scale (see figure 6(A)) [11, 12, 23–37]. As illustrated, research efforts towards miniaturisation have been focused on actuators with bending and extending deformations, due to a lack of advanced manufacturing techniques that are needed to create other more complex deformations [38, 39]. Actuators with other deformation modes are typically



**Figure 6.** (A) Actuation speed vs size plot of soft inflatable actuators from published studies benchmarked against the actuators developed in our labs. The mode of deformation of the actuators is classified with colour codes. (B) We demonstrate the speed of our soft actuators via the built of a flycatcher. The design is based on a 3D-printed base (black part) and three bending-type actuators that catch the fly (*Lucilia cuprina*). (C) Pre-programmed transformation of linear soft actuators into the Queensland University of Technology (QUT) logo. (D) Two actuators with a diameter of 10.5 mm (left) and 600  $\mu$ m (right) demonstrating the capability of our methodology to fabricate actuators with different dimensions. (E) An endoscopic device consisting of three elongation-type soft actuators embedded with a fibre optic cable navigating within the network of 3 mm  $\times$  3 mm channels.

found at larger scales [15, 40–43], where limited dynamic data is provided for them. Whereas we showed that additive manufacturing can create pathways for high dynamic actuators, typically these technologies are not well suited for miniaturisations or combining multiple materials [40, 44–49]. Therefore, as shown in figure 6(A), the majority of the miniaturised actuators were fabricated via various moulding techniques and are limited to only one actuation mode . As demonstrated, our MEW-based fabrication method provides large design flexibility, giving rise to an increase in the design space of miniaturised actuators. We identified

that our actuators are not only among the fastest in their class (actuators based on forced inflation) but also outperform their counterparts with respect to achievable deformation diversity (figure 6(A)). Our mechanical tests indicate that our bending actuators are able to exert similar contact forces when compared to a previously reported actuator design [39], yet they require approximately 5- to 10-times less air pressure. Furthermore, the collected data also points to the same conclusion as Nature-smaller systems are able to reach higher actuation speeds.

Finally, to demonstrate the performance of our actuators, we applied them in a setting where speed, small scale and compliance are of the utmost importance: catching a fly without killing it. Towards this goal, we developed a soft robotic flycatcher (figure 6(B)). The flycatcher consists of three bending actuators that are placed in a triangular pattern around a 3D-printed base with a central cylindrical target area (see supplementary text for further information and figure 9 for the technical drawing). When a fly is detected, we apply air pressure to the three bending actuators using a syringe, rapidly closing the trap, successfully catching the fly in the process (see figure 6(B) and movie S11). Furthermore, by combining various actuator transform into very complex shapes (figures 6(C) and (D) and movie S12) and endoscopic systems that are able to navigate through complex and constrained environments (figures 6(E) and S10 and movie S13) can be developed.

### 3. Conclusions

In conclusion, by enabling the seamless implementation of the concepts of fibre-reinforcement to control volumetric expansion and miniaturisation, we were able to create a wide variety of actuator deformations with the application of only a few microliters of actuation volume. As volume flux is typically the limiting factor for speed, we were able to create high dynamic motions (up to  $\sim$ 30 Hz) using standard pressure regulators. Our additive manufacturing-based automated manufacturing platform allowed us to down-scaling the dimensions of fibre-reinforced actuators without compromising their functionality. Such actuators that operate with low-volume and -pressure fluids and exhibit minor volumetric changes are also highly advantageous in applications where space of operation is limited. Arrays of these actuators may be useful in locomotion, fluid transportation, propelling and mixing, as well as complex grasping and manipulation tasks. In the present study, we focused our investigation on inflatable elastomers. Yet, in future studies, alternative actuation methods can be explored by incorporating different matrix materials that respond to alterations in osmotic conditions, pH, magnetic fields or temperatures. The deformation of this type of soft matter can be guided via our 3D-printed fibrous network, which may unlock new research directions towards the development of a new generation of soft smart materials, actuators and robots.

#### 4. Experimental

#### 4.1. Fabrication of soft actuators

The fabrication process of the actuators starts with the application of a viscous platinum-catalysed silicone (Ecoflex 00–10, Smooth-On Inc., USA) onto the printing collector (rod) of the MEW device. First, parts A and B of the silicon were mixed and stirred thoroughly for 2 min (1A:1B by volume), drawn into a positive displacement pipette and applied to the rotating printing collector after 18 min until a volume sufficient for a wall thickness of ~200  $\mu$ m is deposited. With the aid of a 0.5 mm diameter metal rod attached to the print head of the MEW system that lightly touches the rotating printing collector, the partially-cured silicon was uniformly dispersed by moving the translational linear stage of the system back and forth along the main axis of the printing collector for ~3 min while it was rotating. Subsequently, the printing process of the fibrous network onto the partially-cured silicon was commenced (see movie S1 for the fabrication process).

For the preparation of the MEW device, first, medical-grade polycaprolactone pellets (Purasorb PC 12, Purac Biomaterials, the Netherlands) were placed in a syringe and heated to a temperature of 80 °C in the extrusion head of the device. After allowing the polymer to reach a steady molten state ( $\sim$ 10 mins), air pressure of 1.5 bar was applied to the syringe using an electro-pneumatic pressure regulator (ITV0030, SMC, Japan) to extrude the molten polymer through a 23 G needle. During the extrusion, a voltage of 4.9–5 kV was applied to the needle that leads to the formation of a fine, stable polymeric jet. The printing collector-to-needle distance was set to 3 mm, and the jet was deposited onto the collector at a speed of 165 mm min<sup>-1</sup> (combined translational (linear stage) and tangential speed (rotational stage)). MEW is a computer-aided manufacturing technique that utilises programs written in G-Code-based numerical controlling language (see table S2). The fibrous networks were printed onto the partially-cured silicone to enhance the adhesion between the matrix material and fibres. After the completion of the printing process, the silicone was left to fully cure for 2 h. The actuators were then peeled off the rods with ethanol aiding as a

lubricant. The actuators were mounted to a pressure source after sealing their tip with air-curing silicone (Sil-Poxy, Smooth-On Inc., USA).

#### 4.2. Characterisation of the soft pneumatic actuators

#### 4.2.1. SEM

The micrographs of the actuators and fibrous networks were acquired using a Tescan MIRA3 SEM. Samples were first gold-coated for 75 s at 30 mA (Leica EM-SCD005 gold sputter coater, Wetzlar, Germany) before imaging.

#### 4.2.2. Characterization of the movement in response to pressure

The actuators were driven with air using a custom-made syringe pump, and their static images, as well as their videos, were acquired using a handheld digital microscope (Dino-Lite Edge 5MP, AnMo Electronics Corporation). The pressure values within the system were measured using a pressure sensor (HSCDANN005PGAA5, Honeywell Sensing and Productivity Solutions), which is placed next to the actuators.

#### 4.2.3. High-frequency and rapid actuation tests (dynamic)

For the high-frequency actuation tests, a pneumatic system (Performus VII, Nordson Electron Fusion Devices, Inc) controlled by an external microcontroller was used. The pressure of the delivered air was adjusted for each actuator type to enable their full range of motion within the duration of each actuation cycle (see figure S7). At higher frequencies ( $>\sim$ 10 Hz), high-pressure values exceeding the regular actuation requirements of the actuators was set as the response of the system was found to be slow to reach the desired levels in the given time if high-pressure values are not used. The pressure values were measured using a pressure sensor (HSCDANN005PGAA5, Honeywell Sensing and Productivity Solutions), which was placed at the actuator-end of the experimental setup.

#### 4.2.4. Measurement of the volumetric expansion (static)

The volumetric expansion of the actuators after their actuation was quantified by measuring the amount of fluid  $(H_2O)$  that is delivered with a positive displacement pipette. The actuators were submerged in water during the tests to avoid the deformations caused by the weight of the supplied water.

#### 4.2.5. Simulations

The response of our actuators upon inflation was modelled using the FE method, using the commercial package ABAQUS (2019/Standard). In the fibre actuator analyses, the silicone rubber was modelled as an incompressible Neo-Hookean hyperelastic material model with an initial shear modulus  $\mu_0$  of 30 kPa. The fibres were modelled as a linear elastic material with Young's modulus *E* of 320 MPa and a Poisson ratio  $\nu$  of 0.3. The cylindrical rubber tube was discretised using 3D tetrahedral hybrid solid elements (element code C3D10H), while the fibres were discretised using three-node quadratic beam elements (element code B32). The fibres and cylindrical tube are meshed separately and connected using a tie constraint that allows no slipping of the fibres relative to the tube. We pressurise the actuators by supplying incompressible fluid to the internal cavity while monitoring the pressure inside and simulate the quasi-static behaviour using a static solver.

#### Data availability statement

All data that support the findings of this study are included within the article (and any supplementary files).

#### Acknowledgments

The authors thank Mr Pawel Mieszczanek and Mr Aaron Foster for their assistance in the experiments. The authors acknowledge the Central Analytical Research Facility of the Institute for Future Environments at QUT. This work was financially supported by the Australian Research Council Industrial Transformation Training Centre (external ref: IC160100026, Queensland University of Technology) and by the Hans Fischer Senior Fellowship at the Institute for Advanced Study, Technische Universitaet München.

#### Author contributions

O B initiated and conceived the research, performed experiments, analysed data and drafted the manuscript; B G performed the simulations, contributed to the design of the study, interpretation of the data and writing

of the manuscript; S L contributed to the design, fabrication and characterisation of the soft actuators; T S performed SEM imaging; K B and D WH provided direction and edited the manuscript.

#### **Conflict of interest**

The authors declare no competing interests.

#### Supplementary information

The online version contains supplementary material.

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## **Supporting Information**

## Ultrafast, Miniature Soft Actuators

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## Characterisation of printing accuracy

The deposition accuracy of the fibres, as well as the wall thickness of the silicon body of the actuators, were evaluated an image processing software ImageJ (National Institutes of Health, USA) on scanning electron microscopy micrographs. The distance between the helical fibres (pitch size) of bending, twinning and elongation-type actuators was measured. Arc angle between the longitudinal places fibres of contraction type (from cross-section views) was measured.

## Mechanical characterisation of the soft actuators

The performance (bending, pulling, holding, pushing, contraction force) of the actuators were characterised using Instron MicroTester (Instron, Australia) equipped with 5N load cell. The displacement rate of 1mm/sec was applied holding force of bending and twining actuators.

### Design and development of the flycatcher

For the fabrication of the flycatcher, three bending-type actuators were placed in a triangular pattern around a 3D-printed base (Polylactic acid) (see **Figure S9** for the technical drawing of the flycatcher). All the actuators were connected to a single syringe via tubing and activated all together by pushing the piston of a syringe. We used meat to attract a fly to the centre of the flycatcher.

#### Finite element analyses

The responses of all actuators upon inflation was modelled using the Finite Element (FE) method, using the commercial package ABAQUS (2019/Standard). Three different type of simulations were run throughout the research:

- The eccentric void actuators (see **Figure 1**), was discretized using second-order hybrid tetrahedral elements (C3D10H). The material was modelled as an incompressible Arruda-Boyce rubber with an initial shear modulus of 30 kPa and a locking stretch  $\lambda_m$  of 3.9. Geometrical parameters were taken from [Ref 11]
- The corrugated membrane actuators (see **Figure 1**) consist of an elastomeric part that is bonded to a layer of paper. Geometrical and material parameters were taken from [Ref 12]. The elastomer discretized using second-order hybrid tetrahedral elements (C3D10H), the paper using triangular thin shell elements (STRI65).
- All fibre-reinforced actuators in the manuscripts consist out of an elastomeric tube surrounding by strain-limiting fibres. we have discretised the cylinder using second-order hybrid tetrahedral elements (C3D10H), while the fibres were discretised using 3-node quadratic beam elements (B32). We modelled the material of the cylinder as neo-Hookean, while the fibres are assumed to be linear elastic with a Poisson's ratio of 0.3. Regarding the fibre-reinforced actuator of Figure 1, the geometrical and material parameters were taken from [Ref 13]. Regarding the fibre-reinforced actuators of Figure 2, we have used the geometrical parameters as depicted in Figure S1, and varied the material parameters between 7.5kPa<μ₀<120kPa for the matrix and 80Mpa<E<1.28Gpa for the fibres.</li>

## Writing QUT logo

To show the versatility of our fabrication process, and the fact that fibre arrangements do not have to be constant over the actuator length, we designed three linear actuators that transform into the shape of "QUT" (the acronym of the Queensland University of Technology) when actuated (**Movie S12**). By segmenting the letters into zones of constant deformation, we can convert the intricate shapes into an axial combination of archetypical deformations. We designed each segment independently to reach its final deformation at a certain pressure that is fixed along all connected segments. This is done using finite element modelling where a scripted program is used to a cycle of different fibre arrangements. The resulting fibre arrangement is then axially stitched together and printed using our MEW process. When pressurizing the three actuators, we indeed see the three letters (QUT) emerging.

### Endoscopic device

By axially combining three elongating actuators into a monolithic structure, we end up with a 3 DOF (2DOF bending + extension) soft robot that is able to navigate through complex and constrained environments (**Figure S10** and **Movie S13**). While the general motion of this robot is determined by a global pushing movement, the actuators are used for steering. As our production process is compatible with medical-grade rubbers and fibre materials, we can envision using this robot for catheterisations and other minimally invasive operations. **Movie S13** show the experimental validation of this system in a 3D constrained environment. By selectively inflating one, two or all supply channels in coordination, we can steer the robot throughout a mock-up setup, and reach all the different branches rapidly.

This endoscopic device was assembled by glueing three miniaturised version (inner diameter 800  $\mu$ m instead of 1 mm) of the elongation-type soft actuators described in the previous sections using a silicone rubber (Sil-Poxy<sup>TM</sup>, Smooth-On Inc., USA). A fibre optic cable was placed at the centre of these actuators. Each actuator in the assembly was connected to a separate microtubing and independently activated using a syringe. For the development of the mock setup, several layers of acrylic sheets were laser-cut and glued together to attain network of 3 mm x 3 mm square channels.



Figure S1. Technical drawing of bending actuators (units mm).



Figure S2. Technical drawing of twining actuators (units mm).



Figure S3. Technical drawing of elongation actuators (units mm).



Figure S4. Technical drawing of contraction actuators (units mm).



Figure S5. Bending actuator without helical fibres exhibiting large parasitic deformations



**Figure S6.** Deformation behaviour and actuation performance of various composite soft actuators. (**A-D**) Representative twining-, elongation- and contraction-type actuators with their scanning electron microscopy images as well as their simulated deformation behaviour. Graphs of inflation volume vs deformation and input pressure vs deformation obtained via experiments (n=3) and simulations are also shown.



**Figure S7.** Air pressure values measured while characterising the actuation performance of the actuators at different actuation frequencies (5, 10, 20 to 30 Hz). At frequencies > ~10 Hz, pressure values exceeding the regular actuation pressure requirements of the actuators were applied to be able to reach the desired pressure levels within each actuation cycle. The duty cycle was kept constant (on-to-off duration ratio of 2) for all the experiments.



Figure S8. Two bending-type soft actuators with a diameter of (A) 600µm and (B) 10.5mm.



Figure S9. Technical drawing of the flycatcher.



Figure S10. Technical drawing of the endoscopic device.

Actuator Type	Fibre Distance (helical fibres) (µm)		Silicone Wall Thickness (µm)		
	Target	Fabricated	Target	Fabricated	
		(n=10)		(n=6)	
Bending	390	$380.12 \pm 11.78$	200	$206.57 \pm 4.28$	
Twinning	200	$192.05 \pm 10.04$	200	$187.62 \pm 7.88$	
Elongation	200	192.13 ± 7.91	200	212.27 ± 4.72	
	Arc angle (longitudinal fibres) (°)		Silicone Wall Thickness (µm)		
	Target	Fabricated	Target	Fabricated	
		(n=10)		(n=6)	
Contraction	36	$36.08 \pm 1.41$	200	$210.57\pm6.53$	

Table S1. Characterisatio	on of the fabricated soft actuators
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 Table S2. GCodes used for the additive manufacturing of the actuators

	ARCHETYPICAL ACTUATORS			
OPERATIONS	BENDING	TWINING	ELONGATION	CONTRACTION
INITIALISATION	G18 G91	G18 G91	G18 G91	G18 G91
SILICONE	F375	F375	F375	F375
SPREADING	G1 Y31 X15	G1 Y31 X15	G1 Y31 X15	G1 Y31 X15
	M98 P1250 L5	M98 P1250 L5	M98 P1250 L5	M98 P1250 L5
	G92 Z0	G92 Z0	G92 Z0	G1 Y-5
	G1 Y-5	G1 Y-5	G1 Y-5	
<b>RETURN TO</b>	G90	G90	G90	G90
PRINT SETUP	G1 X0 Y0	G1 X0 Y0	G1 X0 Y0	G1 X0 Y0
	F165	F165	F165	F165
	G91	G91	G91	G91
			G92 Z0	
PRINTING THE	M98 P1266 L3	M98 P1266 L3	M98 P1266 L3	G1 X15 Z240
FIBROUS	G90	G90	G1 Y15	G92 Z0
NETWORKS	G1 X0	G1 X0	G90	M98 P1266 L6
	M30	M30	G1 X0	G90
			G91	G1 X0
			M30	M30
FIBRE PRINT	O1266	O1266	O1266	O1266
LOOP SUB-	G90	G1 X65 Z1040	G1 X65 Z1040	G1 X50
PROGRAM	F165	G1 X1	G1 Y20	G1 Z0.3142
	G1 X-5 Z50	F500	F500	G1 X-50
	G1 X0 Z0	G1 X6 Z-1040	G1 X-65 Z-	G1 Z0.3142
		F165	1040	G1 X50
	G1 X75 Z600	G90	F165	G1 Z0.3142

G1 Y23	G1 X15	G1 Y-20	G1 X-50
G1 Z650	G92 Z0	G1 X-2	G1 Z0.3142
G1 X0	G1 Z 0.2	G1 X2	G1 X50
G1 Y0	G1 X66	M99	G1 Z0.3142
G1 Z600	G1 Z-0.2		G1 X-50
	G1 X0		G1 Z0.3142
G1 X-5 Z650	G1 Z0		G1 X50
G1 X0 Z600	G91		G1 Z0.3142
	M99		G1 X-50
G1 X75 Z0			G1 Z0.3142
G1 Z0.5			G1 X50
G1 X1			G1 Z0.3142
G1 X0			01200112
G1 Z0 7			G1 X-1 5
G1 X75			01 11 1.5
G1 70 3			G1 X-1 5 Z-
G1 X0			25 136
G1 Z0			G1 Z-12 568
F165			G1 X-4
G91			G4 P0 5
M99			G1 Z12 568
10177			G1 X-4
			G4 P0 5
			G1 7-12 568
			G1 X-1 5 7-
			25 136
			25.150
			G1 X-1
			01 / -1
			G1 X-1 5 7-
			25 136
			23.130 G1 7-12 568
			G1 X-4
			$G_{4}$ P0 5
			G1 712 568
			$G1 X_{-4}$
			$G_{4}$ P0 5
			G1 7-12 568
			G1 X-1 5 7-
			25 136
			25.150
			G1 X-1
			G1 X-1 5 7-
			25 136
			G1 Z-12 568
			G1 X-A
			$G_{1} \xrightarrow{\Lambda_{-+}} G_{4} P_{0} 5$
			G1 712 568
			G1 X-4
			~ 1 1 1

				G4 P0.5
				G1 Z-12.568
				G1 X-1.5 Z-
				25.136
				G1 X-1
				G1 X-1.5 Z-
				25.136
				G1 Z-12.568
				G1 X-4
				G4 P0.5
				G1 Z12.568
				G1 X-4
				G4 P0.5
				G1 Z-12.568
				G1 X-1.5 Z-
				25.136
				G1 X-1.5
				G90
				G1 X0 Z0
				G1 X15
				G91
				M99
SILICONE	O1250	O1250	O1250	O1250
SPREADING	G1 X50 Z-100	G1 X50 Z-100	G1 X50 Z-100	G1 X50 Z-100
LOOP SUB-	G1 X-50 Z-100	G1 X-50 Z-100	G1 X-50 Z-100	G1 X-50 Z-100
PROGRAM	M99	M99	M99	M99